



**INTERNATIONAL JOURNAL OF ENGINEERING SCIENCES & RESEARCH
TECHNOLOGY**

**A Performance of a Window Air Conditioner Using Alternative Refrigerants R22
AND R410A**

Abhishek Kumar^{*1}, R.C.Gupta²

^{*1,2} Department of Mechanical Engineering, Jabalpur Engineering college, Jabalpur, India
abhishek_megec@yahoo.com

Abstract

This paper presents the experimental performance analysis of a window air conditioner using two refrigerants R22 and R410A. The effect of the different parameters of performance analysis (refrigeration capacity, COP, compressor power, pressure ratio) were investigated for various evaporating temperature and ambient temperature. The result shows that refrigerant R22 is better than R410A in case of COP, refrigeration capacity but for pressure ratio and compressor power R410A shows better performance than R22. Since R22 refrigerant has better performance than also this refrigerant is replaced by the end of 2015. Due to their severe environmental impact.

NOMENCLATURE:

COP : Coefficient of Performance
GWP : Global Warming Potential
M : Mass Flow Rate
PR : Pressure Ratio
T : Temperature
EER : Energy Efficiency Ratio
HCFC : HydrochloroFluro Carbon

ODP: Ozone Depletion Potential
h : Specific Enthalpy (KJ/KG)
P: Pressure (KPA)
Q : Heat Transfer Rate (KW)
Th: Thermometer
HC: Hydrocarbon
T_{Cond.}: Condenser Temperature (K)

Keywords: Window air conditioner, R22, R410A, Azeotropic mixture

Introduction

During the last decade, the number of refrigerants used in refrigerating unit has dramatically increase as a consequence of the elimination of the CFC'S and HCFC'S. Recently the ozone depleting potential (ODP) and global warming potential (GWP) have become the most important criteria in the development of new refrigerant apart from the refrigerant CFC's and HCFC's. In spite of the high GWP alternative to refrigerants CFC's and HCFC's such as hydrofluorocarbon (HFC) refrigerants with their zero ODP have been preferred for use in many industrial and domestic applications intensively for decades. HFC refrigerant also have suitable specification such as non-flammability, stability and similar vapour pressure to the refrigerant CFC's and HCFC's. R22 is one of the important refrigerants used in air conditioning all over the world. R22 is controlled substance under the montreal protocol. It has to be totally phased out by 2015. In Europe, HCFC's already have been phased out in 2002, and the total phased out of HCFC's is scheduled in 2015. R22 replacement option for air conditioner, heat pump, and refrigeration system can be grouped in

three categories are fluorocarbons, that are used in conventional vapour compression cycle such as R134a, R410A, R407C, alternative fluid which include propane R290 and R717 and are also used in vapour compression cycle, and finally alternative cycles that include absorption systems and use trans critical fluids (CO₂) and air cycle. In general these alternative technologies do not currently offer the same energy efficiency as the vapour compression cycle.

The HFC refrigerant are considered as one of the fix target green house gases under the Kyoto Protocol Of United Nation Frame Work Convention on climate change (UNFCCC). In 1997 {31,32} Kyoto Protocol was approved by many nation called for the reduction in emission of green house gas including HFC refrigerants. The presence of fluorine atoms in R134A is responsible for the major environmental impact (GWP) with serious implication for the future development of the refrigeration based industries.

The refrigerant R410A is a near azeotropic blend of R32 and R125 with a critical temperature of 72.8 °C and a critical pressure of 4.86 MPa. Its ozone

depletion potential is zero and it has been adopted in air conditioning and heat pump system for residential application. R410A has high volumetric cooling capacity, which means that this refrigerant can absorb significant amount of heat from the air for a unit volume of refrigerant in a direct expansion evaporator. R410A operates at high pressures than R22 and its GWP is 2,088 [33]. Several researcher investigated refrigerants that could potentially retrofit R410A in air conditioning system. In India, about one million room air conditioner (ACs) are manufactured in various capacities every year with refrigerant R22, which seriously depleted ozone layer and contribute to green house effect. The substitute for refrigerant R22 is R410A because of its close match to R22 in existing window air conditioner. R410A can be directly used in place of R22 compressor with little modification. Polyol Ester (POE) is to be used with R410A instead of mineral oil.

Experimental Setup

The schematic diagram representing the air conditioner is shown in figures 1, 2. The unit was retrofitted with R22 and R410A. In order to have a uniform temperature throughout the room, ceiling fan of 60 watt power installed in the centre of room was used to circulate the air inside the room.

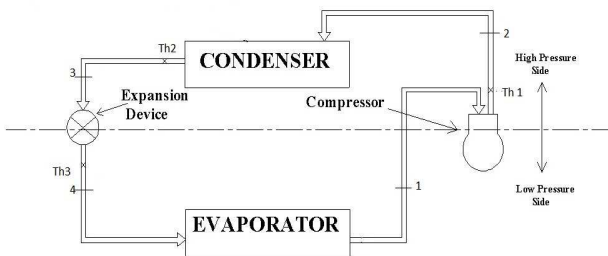
To measure compressor power, a multimeter with ±0.5 % accuracy was used. Room temperature was measured with the help of precision thermometer with an accuracy of ±0.01°C. The specification of air conditioner test unit is given in table no. 1.



FIG:2a –Open Unit FIG:2b –Closed Unit
Fig:2-Air Conditioner Experimental Test Unit

Table:1 : Specification Of Air Conditioner Test Unit

COMPRESSOR:	=	Reciprocating Type
FAN:	=	2900 RPM
CAPILLARY:	=	Cu Tube
EVAPORATOR:	=	Cu Coil
TEMPERATURE MEASURING RANGE	=	-50°C TO 80°C(accuracy±1°C)
RELATIVE HUMIDITY RANGE	=	20% TO 85%
CAPILLARY LENGTH	=	832 mm
EVAPORATOR:AVG SAT. TEMP.	=	10°C to 12°C
CONDENSER:AVG SAT. TEMP.	=	40°



Where: Th: thermometer
Fig 1: Experimental Unit Cycle

Performance Parameters Analysis

The equations for the cycle analysis can be obtained by means of mass and energy conservation. The data reduction of the theoretical results can be analysed below. The pressure ratio of the cycle can be seen below as follows:

The pressure ratio = $P_{cond.} / P_{evap.}$

The refrigerating effect (RE), in other words, the heat transfer rate of the evaporator ($Q_{evap.}$) is calculated as follows:

$R_E = Q_{evap} = h_1 - h_4$ kJ/kg

Isentropic compression work of the compressor (W_{comp}) is expressed as follows:

$W_{comp} = h_2 - h_1$ kJ/min

The coefficient of performance (COP) of the refrigeration system's cycle can be determined by:

$COP = R_E / W_{comp}$

Power of refrigeration is calculated as follows:

Power piston of refrigerant = $W_{comp} / 60$ kw

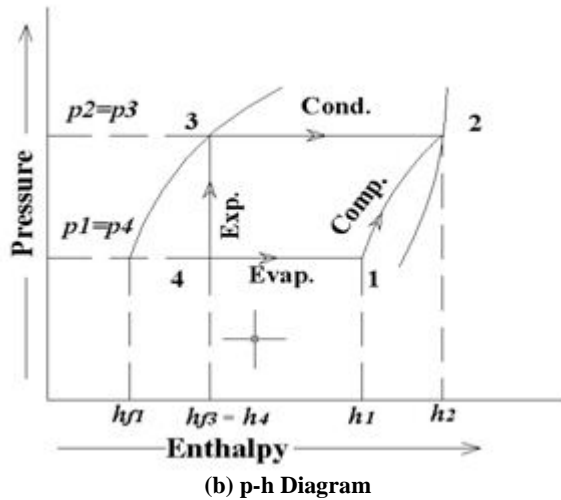
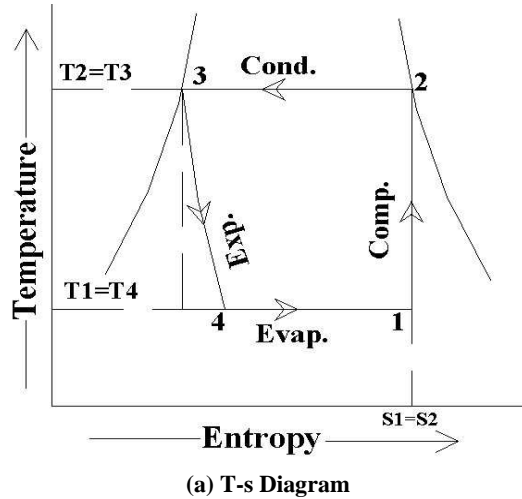


fig 2: Theoretical vapour Compression Cycle

PROPERTIES	REFRIGERENT	
	R 22	R 410 A
Composition	CHClF ₂	50+5,-1.5% CH ₂ F ₂ 50+1.5,-.5% C ₂ HF ₅
Critical Temp. (°C)	96.14	70.5
Critical Pressure (kPa)	4990	4770
ODP	0.05	0
GWP(100years)	1810	2100
Boiling Temp.(°C)	-40.8	-51.4
Atm life(yrs)	96.1	18.155
MOLECULER WT. (g/mol)	86.5	72.6

Result and Discussion

R22 and its retrofit refrigerant R410A were used in window air conditioner and system performance were evaluated and compared. The result of the refrigeration capacity obtained at different evaporating temperature is shown in figure:3. Evaporating temperature varied from 2°C to 12°C as a result of the variation of the indoor temperature from 17°C to 26°C using the system temperature control. It was observed that for all the investigated refrigerants, the refrigeration capacity increased with increase in evaporating temperature. At the same time, evaporating temperature for refrigeration capacity obtained with the R22 system is higher than that from the R410A system. Average refrigeration capacity of R410A is 14.1% lower than that with R22.

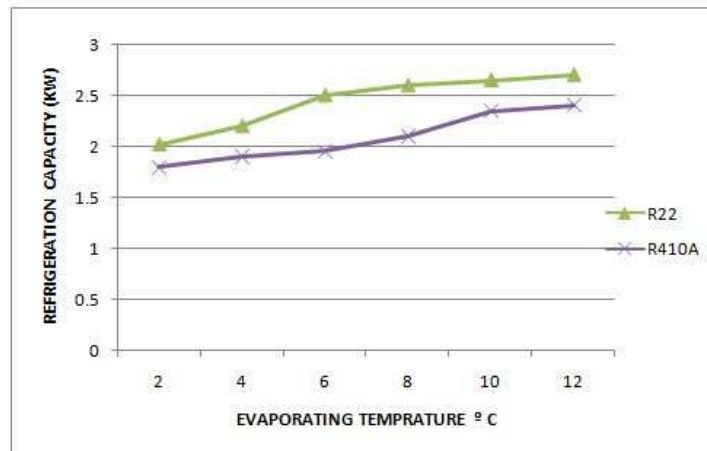


Figure 3: Variation of refrigeration capacity with evaporating temperature

Figure:4 Shows the variation between the compressor power and the evaporating temperature of R22 and the alternative refrigerants R410A. As shown in figure, the change of compressor power with evaporating temperature is similar for both the refrigerants. As the

evaporating temperature is increase the compressor power also increases. The compressor power with R410A is higher than those with R22. The average compressor power with R410A increased by 11%.as compared with R22.

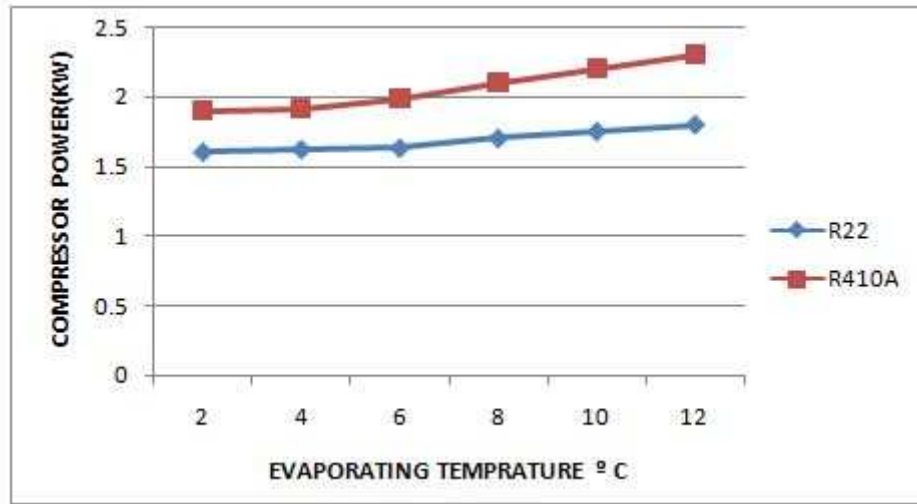


Figure 4: Variation of compressor power capacity with evaporating temperature

The performances of investigated refrigerants in window air conditioner were obtained for different ambient air temperature. The ambient air temperature varied from 25°C in the early hours of day to 40°C in the late afternoon. The refrigeration capacity and COP obtained

with R22 and R410A at various ambient temperature is plotted in figure:6 and figure:7. As shown in these figures, the refrigeration capacity and COP reduce as ambient air temperature increase. Also it can be seen from these figures, that the performance with R22 is better than R410A.

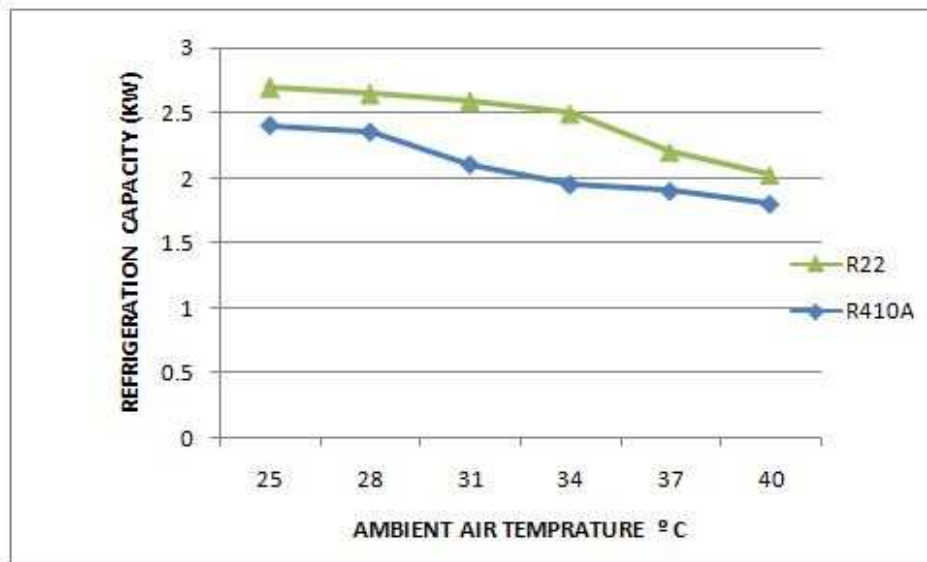


Figure 6 :Effect of ambient air temperature on the refrigeration capacity

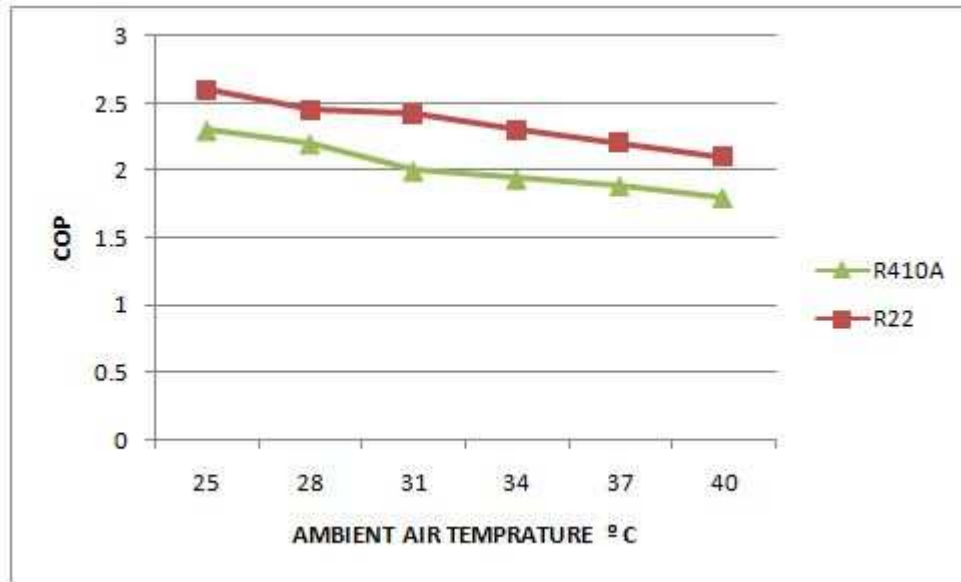


Figure 7: Effect of ambient air temperature on the coefficient of performance (COP)

Figure:8 Compare the variation of compressor power for the two refrigerants in terms of ambient temperature as shown in these figures. The compressor power increase as the ambient temperature increases but

there is considerable difference in the performance with R22 and R410A. The compressor power with R410A were found to be highest among the R22 refrigerant at all ambient air temperature.

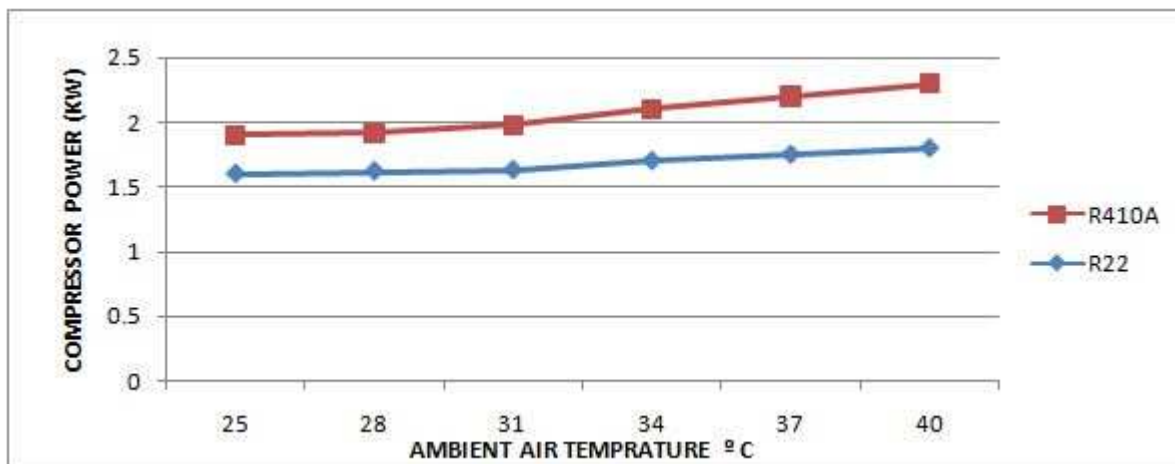


Figure 8: Effect of ambient air temperature on the compressor power

Figure:9compare the variation of pressure ratio for the two refrigerants in terms of ambient temperature as shown in these figures. The pressure ratio increase as the ambient temperature increases but there is considerable difference in the performance with R22 and R410A. The pressure ratio with R410A were found to be

highest among the R22 refrigerant at all ambient air temperature.

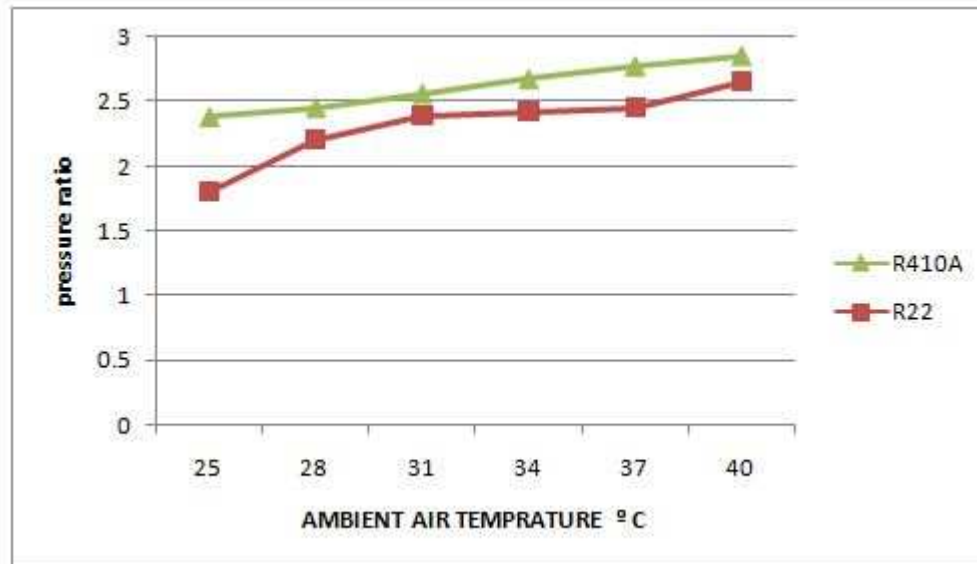


Figure 9: Effect of ambient air temperature on the pressure ratio

Conclusions

In this study, experiments were carried out to investigate R22 and its retrofit substitute R410A in a window air conditioner. Based upon experimental result, the following conclusions were drawn

1. The refrigerant capacity and COP reduce and compressor power and pressure ratio increase in case of R410A.
2. The performance parameters i.e. refrigeration capacity, COP increases with increase in evaporating temperature in case of R22.
3. The compressor power of R410A is higher than R22.
4. The average COP of R410 is lower than the R22.

Finally, the system when charge with R22 consistently had the best performance when compared with system containing R410A. But then also the refrigerant R22 is replaced by R410A because of its high ozone depletion which severely affects our environment. R410A is zero ozone depletion and high volumetric cooling capacity. R410A refrigerant operates at high pressure than R22.

References

- [1] M. PAULUS- LANCKRIET, O. BUYLE, R407C/R410A :an analysis of the two prominent candidate for the replacement of R22 in refrigeration application ,in proceeding of IIF –IIR conference , Linz ,Austria ,1997, PP 85-93.
- [2] D LEE .VAHNA,Y.KIM ,Y.CHANGA ,L.NAM, Experimental investigation on the

drop in performance of R407C as a substitute for R22 in a screw chiller with shell and tube heat exchanger .International journal of refrigeration 25(2001) 575-585.

- [3] C.APREA, AGRECO: An experimental evolution of the green house effect in R22 substitution, Energy conversion and management 39(1998) 877-887.
- [4] C.APREA, A.GRECO,: An exetetic analysis of R22 substitution ,Applied thermal engg. 22 (2002) 1455-1469.
- [5] C.APREA, A GRECO, Performance evolution of R22 and R407c in a vapour compression , Applied Thermal Engg. 23 (2003) 215-227.
- [6] R.CABELLA, E.TORRELLA, J. NAVARRO – ESBRI, Experimentalevolution of a vapour compression plant performance using R134A, R407C and R22 as working fluids ,Applied Thermal Engg. 24 (2004) 1905-1917.
- [7] S.DEVOTTA , AS PADALKAR , N.K.SANE ,Performance assessment of HCFC 22 window air conditioner retrofitted with R407 C ,Applied Thermal Engg . 25 (2005) 2937 -2949.
- [8] H. KONG, Comparison of R410a , R407c and propane in heat pump application in proceeding of IIF-IIR conference ,Linz Austria 1997 ,PP 94-103.
- [9] M.W.SPATZ ,Y.MOTTA , AN evolution of option for replacing HCFC-22 in medium temperature refrigeration systems , International Journal of Refrigeration 27 (2004) 475-483.

- [10] C.APREA , C.RENNO ,Experimental comparison of R22 with R417A performance in a vapour compression refrigeration plant subjected to a cold store , Energy Conversion and Management 45 (2004) 1807-1819.
- [11] M.YILMAZ, Performance analysis of a vapour compression heat pump using Zeotropic refrigerant mixture, Energy Conversion and Management 44(2002)267-282.
- [12] A.S.DALKILIC,AND S. WONGWIRE ,” A performance comparison of vapour compression refrigeration system using various alternative refrigerant” International Communication in Heat and Mass Transfer 37,pp 1340-1349.
- [13] DONGSOO JUNG,, BONGJIN PARK, HYUNCHUL LEE, Evaluation of supplementary/retro@refrigerants for automobile air-conditioners charged with CFC12 .(1999)558-568
- [14] J.D. Douglas, J.E. Braun, E.A. Groll, D.R. TREE ,A cost-based method for comparing alternative refrigerants applied to R-22 systems . (1999) 107–125
- [15] S. Devotta, A.V. Waghmare , N.N. Sawant , B.M. Domkundwar, Alternatives to HCFC-22 for air conditioners(2001) 703-715
- [16] MAN –HOE KIM, CLARK W.BULLARD, Dynamic characteristics of a R410A split air – conditioning system.(2001)652-659
- [17] S.DEVOTTA, A.S.PADALKAR ,N.K.SANE, Performance assessment of HCFC 22 window air conditioner retrofitted with R407C(2005)2937-2949
- [18] D.B.JABARAJ, P.AVINASH, D.MOHANLAL, S.RENGANARAYAN, Experimental investigation of HFC 407C/HC290/HC600 mixture in a window air conditioner.(2006)2578-2590
- [19] P.SARNITICHARTSAK, V.MONAYKUL, S.T HEPA, A.NATHAKARANAKULE ,Simulation and experimental evolution of the effects of oil circulation in an inverter air conditioning system using R22 and R407C.(2006)1481-1491
- [20] ZHENG LI, SHIMING DENG, An experimental study on the inert operational characteristics of a direct expansion air conditioning unit.(2007) 1-10
- [21] D.B.JABARAJ, A. NARENDRAN, D.MOHAN LAL, S.RENGANARAYANAN, Evolving an optimal composition of HFC 407C /HC290 /HC600A mixture as a alternative to HCFC22 in window air conditioner(2007)276-283.
- [22] T.S.RAVIKUMAR , D. MOHANLAL ,On road performance analysis of R134A /R600A/ R290 refrigerant mixture in an automobile air conditioning system with mineral oil as a lubricant.(2009) 1891-1901
- [23] M FATOUH ,TALAAT A .IBRAHIM , A. MOSTAFA ,Experimental investigation on a solid desiccant system integrated with a R407c compression air conditioner(2009)2670-2679.
- [24] TOLGA N .AYNUR , Variable flow refrigerant system: a review(2010)1106-1112.
- [25] MFATOUH ,TALAAT A .IBRAHIM , A. MOSTAFA , Performance assessment of direct expansion air conditioner working with R407C as an R22 alternative.(2010)127-133.
- [26] CLAUDIO ZILIO , J. STEVEN BROWN ,GIOVANNI SCHIOCHET , ALBERTO CAVALLINI, The refrigerant R123yf in air conditioning system(2011)6110-6120.
- [27] BUKOLA OLALEKAN BOLAJI , Performance investigation of ozone friendly R404A and R507 refrigerant as alternatives to R22 in a window air conditioner(2011)3139-3143
- [28] CIROAPREA, ANGELO MAIORINO, An experimental investigation of the global environmental impact of the R22 retrofit with R422D (2011)1161-1170.
- [29] C J WU ,D.P.LIU, J.PAN ,A study of aerodynamic and acoustics performance of a indoor unit dc inverter split air conditioning(2012)415-422.
- [30] PONGSAKORN SARNTICHARTSAK ,SIRICHAJ THEPA , Modeling and experimental study on the performance of a an inverter air conditioner using R410a with evaporative cooled condenser.
- [31] JOHNSON 1998, Global warming from HFC, environment impact assessment rev,18, 485 – 492
- [32] WEN-TIENTASI, 2005, an over view of environmental hazards and exposure, and explosive rise of hydrofluorocarbon HFC'S, chemosphere, 61, 1539-47
- [33] SUSAN SOLOMON ,GIAN-KASPER PLATTNER ,RETO KNUTT, FREDLINGSTEIN ,Irreversible climate change due to carbon dioxide emissions [2007]